

Europäisches Patentamt
European Patent Office
Office européen des brevets



(11) **EP 1 275 916 A2**

(12)

EUROPEAN PATENT APPLICATION

(43) Date of publication:
15.01.2003 Bulletin 2003/03

(51) Int Cl.7: **F25B 41/06, F16K 31/68,
G05D 23/12**

(21) Application number: **02254796.2**

(22) Date of filing: **09.07.2002**

(84) Designated Contracting States:
**AT BE BG CH CY CZ DE DK EE ES FI FR GB GR
IE IT LI LU MC NL PT SE SK TR**
Designated Extension States:
AL LT LV MK RO SI

(72) Inventors:
• Yano, Masamichi, Fujikoki Corporation
Tokyo 158-0082 (JP)
• Watari, Daisuke, Fujikoki Corporation
Tokyo 158-0082 (JP)

(30) Priority: **12.07.2001 JP 2001211690
28.12.2001 JP 2001400573**

(74) Representative: **Mouteney, Simon James
MARKS & CLERK,
57-60 Lincoln's Inn Fields
London WC2A 3LS (GB)**

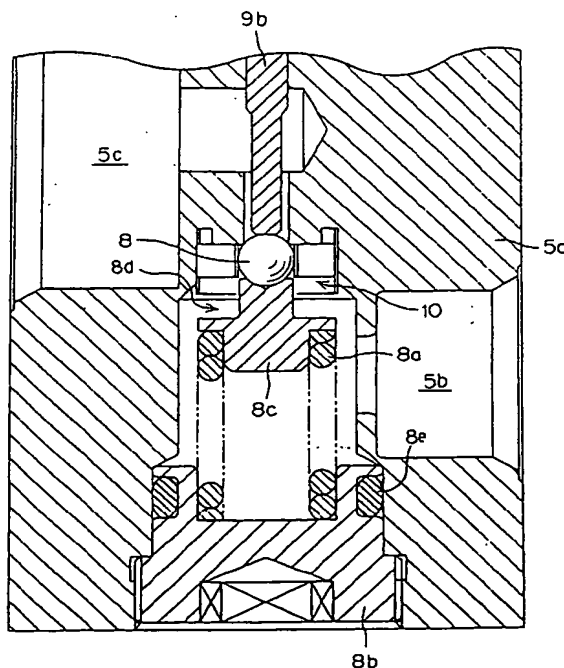
(71) Applicant: **Fujikoki Corporation
Tokyo 158-0082 (JP)**

(54) Expansion valve

(57) The valve body (5a) has an orifice (7) that provides communication between a high-pressure side passage (5b) through which a cooling medium flows in and a low-pressure side passage (5c) through which the cooling medium flows out. Also, the valve (5) is provided with a valve element (8) that adjusts the volume of the cooling medium passing through this orifice, an operat-

ing rod (9b) that operates the valve element (8) in the valve opening position, and a temperature-sensing drive section (9) that drives this operating rod (9b). On the upstream side of the orifice (7) of the high-pressure side passage (5b) is disposed a support ring (10) that constrains the ball-shaped valve element (8) of the valve body (5a).

FIG.1



EP 1 275 916 A2

Description

BACKGROUND OF THE INVENTION

Field of the Invention

[0001] The present invention relates to an expansion valve that constitutes a refrigerating cycle.

Description of the Prior Art

[0002] Although there are various types of expansion valve, widely used is an expansion valve in which a valve element is disposed, from the upstream side, opposite to an orifice which is formed by narrowing a high-pressure cooling medium passage, through which a high-pressure cooling medium to be fed into an evaporator flows, and the valve element is caused to perform opening and closing operation in response to the temperature and pressure of a low-pressure cooling medium discharged from the evaporator.

[0003] An expansion valve of this type can be used in a refrigerating cycle 1 in an air conditioner or the like of an automobile, as shown in FIG. 11. This refrigerating cycle 1 is composed of a cooling medium compressor 2 driven by an engine, a condenser 3 connected to the cooling medium compressor 2 on the discharge side thereof, a receiver 4 connected to the condenser 3, and an expansion valve 5 that causes the liquid-phase cooling medium from the receiver 4 to expand adiabatically so as to convert it into a gas-liquid two-phase cooling medium, and an evaporator 6 connected to the expansion valve 5. The expansion valve 5 is positioned within the refrigerating cycle 1.

[0004] The expansion valve 5 is provided with a high-pressure side passage 5b, through which the liquid-phase cooling medium flows into the valve body 5a, and a low-pressure side passage 5c, through which the gas-liquid two-phase cooling medium that has adiabatically expanded flows out. The high-pressure side passage 5b and low-pressure side passage 5c communicate with each other via an orifice 7. Furthermore, the expansion valve 5 is provided, in a valve chamber 8d thereof, with a valve element 8 that adjusts the volume of the cooling medium passing through the orifice 7.

[0005] A low-pressure cooling medium passage 5d pierces through the valve body 5a of the expansion valve 5. Furthermore, a plunger 9a is slidably disposed within this low-pressure cooling medium passage 5d. This plunger 9a is driven by a temperature-sensing drive section 9 fixed to the upper part of the valve body 5a. The interior of this temperature-sensing drive section 9 is divided by a diaphragm 9d so that an upper airtight chamber 9c and a lower airtight chamber 9c' are formed in the temperature-sensing drive section 9. A disk portion 9e at the top end of the plunger 9a abuts against the diaphragm 9d.

[0006] Furthermore, a compression coil spring 8a,

which presses the valve element 8 via a support member 8c in the valve closing direction, is disposed within the valve chamber 8d in the lower part of the valve body 5a. This valve chamber 8d is blocked by an adjusting screw 8b screwed into the valve body 5a and is held in an airtight condition by an O-ring 8e.

[0007] Also, an operating rod 9b that moves in the valve opening direction by the sliding action of a plunger 9a abuts against the bottom end of the plunger 9a.

[0008] And the plunger 9a in the temperature-sensing drive section 9 transmits the temperature in the low-temperature cooling medium passage 5d to the upper airtight chamber 9c. The pressure of the upper airtight chamber 9c changes in response to the transmitted temperature. For example, when the temperature transmitted to the upper airtight chamber 9c is high, the pressure of the upper airtight chamber 9c increases so that the diagram 9d pushes the plunger 9a down. As a result, the valve element 8 moves in the valve opening direction so that the volume of the cooling medium passing through the orifice 7 increases, whereby the temperature of the evaporator 6 is lowered.

[0009] On the other hand, when the temperature transmitted to the upper airtight chamber 9c is low, the pressure of the upper airtight chamber 9c drops, the force for pushing the plunger 9a down by means of the diagram 9d becomes weak, and the valve element 8 moves in the valve closing direction due to the action of the compression coil spring 8a, which presses the valve element 8 in the valve closing direction, with the result that the volume of the cooling medium passing through the orifice 7 decreases and that the temperature of the evaporator 6 is raised.

[0010] In this manner, according to the temperature change in the low-pressure cooling medium passage 5d, the expansion valve 5 moves the valve element 8 to change the opening area of the orifice 7 and adjust the volume of the cooling medium passing through the orifice 7, thereby adjusting the temperature of the evaporator.

[0011] And in the expansion valve 5 of this type, the relationship between the temperature in the low-pressure cooling medium passage 5d and the opening area of the orifice 7 which causes the liquid-phase cooling medium to expand adiabatically so as to convert it into a gas-liquid two-phase cooling medium can be set by adjusting the spring load of the compression coil spring 8a which presses the valve element 8 in the valve closing direction, by adjusting the screw-in amount of the adjusting screw 8b.

[0012] However, pressure fluctuations in the high-pressure cooling medium fed into the expansion valve may sometimes occur on the upstream side in the refrigerating cycle, and these pressure fluctuations are transmitted to the expansion valve with the high-pressure cooling medium liquid serving as a medium.

[0013] Then, in a conventional expansion valve as described above, when the cooling medium pressure on

the upstream side is transmitted to the valve element by pressure fluctuations, the pressure fluctuations may sometimes pose the problem that the operation of the valve element become unstable. In this case, the flow control of the expansion valve is not accurately performed. Or this may sometimes cause the irregularity that the vibration of the valve element produces noise.

[0014] As a measure to solve this problem, there has been proposed a technique in which a spring gives an urging force sideways to a rod which is disposed so as to freely move forward and backward in an axial direction between a power element and a valve element so that an operation is stabilized (see Japanese Patent Application Laid-Open No. 2001-141335).

[0015] With the conventional technique mentioned above, however, although the purpose of coping with pressure fluctuations of a high-pressure cooling medium for stabilization of operation is achieved, the spring that pushes sideways the rod which moves forward and backward in an axial direction must be arranged in a stable condition, so that there is a fear of requiring high cost because of a complex structure and assembly work.

OBJECTS AND SUMMARY OF THE INVENTION

[0016] The object of the invention is to provide an expansion valve which enables stable operation against pressure fluctuations of a high-pressure cooling medium using simple and inexpensive means.

[0017] In order to achieve the above-described object, the expansion valve of the present invention comprises a valve body which has an orifice that provides communication between a high-pressure side passage through which a cooling medium flows in and a low-pressure side passage through which the cooling medium flows out; a valve element that adjusts the volume of the cooling medium flowing through the orifice; an operating rod that operates the valve element in the valve opening direction; and a temperature-sensing drive section that drives the operating rod. This expansion valve further comprises constraining means for constraining the above-described valve element or for constraining support member that is integral with this valve element, which is disposed on the upstream side of the orifice of the high-pressure side passage.

[0018] In this constraining means it is possible to adopt the following embodiments:

[0019] The constraining means is attached to the above-described valve body.

[0020] The constraining means gives a constraining force to the valve element by an elastic force.

[0021] The valve element is formed in the shape of a ball, and the constraining means is a support ring that supports the valve element.

[0022] The support ring comprises an elastically deformable, annular ring-shaped portion and a vibration-isolating spring. The vibration-isolating spring supports the valve element.

[0023] As the expansion valve of the present invention comprises the above-described components, by disposing constraining means of simple structure for constraining the valve element or valve-element support member, on the upstream side of the orifice, it is possible to suppress the vibration of the valve element caused by pressure fluctuations of the cooling medium on the upstream side of the refrigerating cycle.

BRIEF DESCRIPTION OF THE DRAWINGS

[0024] The foregoing and other objects and features of the invention will become apparent from the following description of preferred embodiments of the invention with reference to the accompanying drawings, in which:

FIG. 1 is a partial sectional view of an expansion valve according to an embodiment of the invention; FIG. 2 is a perspective view of a first example of a support ring used in the expansion valve shown in FIG. 1;

FIG. 3 is a perspective view which shows how the support ring shown in FIG. 2 constrains a valve element;

FIG. 4 is a perspective view of a second example of a support ring used in the expansion valve shown in FIG. 1;

FIG. 5 is a perspective view of a third example of a support ring used in the expansion valve shown in FIG. 1;

FIG. 6 is a perspective view which shows how the support ring shown in FIG. 5 is attached to the expansion valve;

FIG. 7 is a perspective view which shows how the support ring shown in FIG. 6 constrains a valve element;

FIG. 8 is a perspective view of a fourth example of a support ring used in the expansion valve shown in FIG. 1;

FIG. 9 is a perspective view which shows how the support ring shown in FIG. 8 is attached to the expansion valve;

FIG. 10 is a perspective view which shows how the support ring shown in FIG. 9 constrains a valve element; and

FIG. 11 is a sectional view of a conventional expansion valve positioned in a refrigerating cycle.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

[0025] First, an embodiment of the expansion valve according to the invention will be described by referring to a partial sectional view of FIG. 1.

[0026] The expansion valve shown in FIG. 1 is characterized in that the circumference of the valve element 8 of conventional expansion valve 5 shown in FIG. 11 is supported by constraining means 10 of a structure

which will be described later and, therefore, examples of structure of this constraining means will be mainly described here. In the following explanation of the expansion valve shown in FIG. 1, the same reference numerals are used for the elements identical with those of the expansion valve shown in FIG. 11.

[0027] A valve element 8 of an expansion valve 5 is driven by a temperature-sensing drive section 9 that operates in response to the temperature of a low-pressure cooling medium fed from an evaporator 6, so that the flow rate of cooling medium flowing into the evaporator 6 is adjusted. Constraining means 10 (described later) that gives a constraining force to this valve element 8 is fixedly attached in a housing space of a circular section, which is formed in the valve body 5a in close vicinity to the valve elements 8. And, with this constraining means 10, the subject of the invention, i.e., elimination of unstable operation of the valve element due to pressure fluctuations of a high-pressure cooling medium, is achieved.

[0028] A valve body 5a has an orifice 7 that provides communication between a high-pressure side passage 5b through which a cooling medium flows in and a low-pressure side passage 5c through which the cooling medium flows out, both passages being formed in the expansion valve 5. The volume of the cooling medium flowing through this orifice 7 is adjusted by the opening area of the valve element 8.

[0029] The adjustment of the opening area of the orifice by the valve element 8 is performed by the operation of an operating rod 9b that operates the valve element 8 in the valve opening direction and of the temperature-sensing drive section 9 that drives this operating rod 9b.

[0030] On the upstream side of the orifice 7 (or, on the side of the high-pressure side passage 5b), constraining means 10 which constrains the valve element 8 is disposed within a valve chamber 8d. This constraining means 10 is, as described above, attached in the housing space formed in the valve body 5a. Using its elastic force, this constraining means 10 constrains the valve element 8 sideways.

[0031] Incidentally, this constraining means 10 is constructed so as not to impede the operation of adjusting the opening area of the orifice 7 by the valve element 8 even when the constraining means 10 constrains the side surface of the valve element 8.

[0032] The valve element 8 is formed in the shape of a ball and supported by a support member 8c that is integral with the valve element 8. The constraining means 10 comprises a support ring that elastically supports either or both of the valve element 8 or the support element 8c. In the following description, the constraining means 10 is referred to as the support ring. The support ring, which serves as constraining means and will be described below, supports the valve element 8 elastically.

[0033] A first example of the support ring will be de-

scribed by referring to FIGS. 2 and 3.

[0034] The support ring 10 in this example comprises an annular ring-shaped portion 11, which is formed from a material of steel having high metal elasticity, such as stainless steel, and is capable of elastic deformation, and a plurality of, for example, four vibration-isolating springs 12 of curved plate, which are formed by cutting this ring-shaped portion 11 so as to protrude from the ring-shaped portion 11. Each of the four vibration-isolating springs 12 is formed in a curved shape so that the leading end thereof takes on a convex shape protruding toward the center of the ring-shaped portion 11. And these four vibration-isolating springs 12 elastically support the ball-shaped valve element 8 at the circumference thereof, as shown in FIG. 3.

[0035] Furthermore, in the support ring 10, a slit 13 is formed in a part of the ring-shaped portion 11 so that the diameter of the ring-shaped portion 11 can be reduced during mounting in the housing space of the valve body 5a.

[0036] According to the support ring 10 of this structure, when the ring-shaped portion 11 is mounted in the housing space of the valve body 5a, the valve element 8 is supported by the vibration-isolating springs 12 at four places in the circumference. Thus, the support ring 10, which functions as the constraining means of the valve element 8, can stabilize the operation of the valve element 8 even when fluctuations in the cooling medium pressure occur in the refrigerating cycle and hence it is possible to perform accurate control of the flow rate of cooling medium and to prevent the production of noise due to the vibration of the valve element 8.

[0037] A second example of the support ring will be described by referring to FIG. 4.

[0038] A support ring 10a in this example comprises one annular ring-shaped portion 11a and a plurality of vibration-isolating springs 12a of plate, which are disposed on one side of this ring-shaped portion 11a. Incidentally, in the support ring 10a, a slit 13a is also formed in a part of the ring-shaped portion 11a so that the diameter of the ring-shaped portion 11a can be reduced during mounting in the housing space of the valve body 5a, in the same manner as in the case of the support ring 10 of the above-described first example.

[0039] Each of the vibration-isolating springs 12a is formed in a curved shape so that the leading end thereof takes on a convex shape protruding toward the center of the ring-shaped portion 11. The valve element 8 is supported at the circumference thereof by the sides of the leading ends of the vibration-isolating springs 12a. In the support ring 10a of this example, the vibration-isolating springs 12a are formed by cutting the ring-shaped portion 11a so as to protrude from this ring-shaped portion 11a, in the same manner as in the case of the support ring 10 of the first example.

[0040] In the support ring 10a of this structure, it is possible to perform accurate control of the flow rate of cooling medium and to prevent the production of noise

due to the vibration of the valve element 8 when fluctuations in the cooling medium pressure occur in the refrigerating cycle, in the same manner as in the case of the support ring 10 of the first example (FIGS. 2 and 3).

[0041] A third example of the support ring will be described by referring to FIGS. 5 to 7.

[0042] In the support ring 10b of this example, an overlapping portion is formed at the end portion of a plate forming a ring-shaped portion 11b, instead of forming the slit 13, 13a in the ring-shaped portion 11, 11a of the support ring 10, 10a in the above-described first and second examples. As shown in FIG. 5, this overlapping portion is formed by extending a tongue 11b' having a narrow width and a prescribed length from one end of a ring-shaped portion 11b with the same curvature as the ring-shaped portion 11b. On the other hand, a tongue-receiving recess 11b", which guides and supports the tongue 11b' constituting the overlapping portion, is formed at the other end of this ring-shaped portion 11b.

[0043] This tongue-receiving recess 11b" is formed so as to extend in the circumferential direction in the vicinity to the other end of the ring-shaped portion 11b between the upper and lower edge portions. And the depth of the tongue-receiving recess 11b" is provided in a manner such that no gap is formed between the ring-shaped portion 11b and the inner wall of the housing space formed in the valve body 5a when the tongue 11b' of the ring-shaped portion 11b overlaps the tongue-receiving recess 11b" within the housing space. That is, the depth of the tongue-receiving recess 11b" is almost the same as or larger than the thickness of the tongue 11b'.

[0044] In the same manner as in the case of the support ring 10, 10a in the above-described first and second examples, the support ring 10b of this example comprises also an annular ring-shaped portion 11b, which is formed from a material of steel having high metal elasticity, such as stainless steel, and a plurality of, for example, three vibration-isolating springs 12b of curved plate, as shown in FIG. 5, which are formed by cutting this ring-shaped portion 11b so as to protrude from this ring-shaped portion 11b. Each of the vibration-isolating springs 12b is formed in a curved shape so that the leading end thereof takes on a convex shape protruding toward the center of the ring-shaped portion 11b. And these three vibration-isolating springs 12b elastically support the ball-shaped valve element 8 at the circumference thereof, as shown in FIG. 7.

[0045] According to the support ring 10b of this structure, the valve element 8 is supported by the vibration-isolating springs 12b at three places in the circumference, a minimum necessary number of places, when this support ring 10b is fixedly attached in the housing space formed in the valve body 5a. That is, the support ring 10b functions as the constraining means of the valve element 8. As a result, even when fluctuations in the cooling medium pressure occur in the refrigerating cycle, the operation of the valve element 8 can be sta-

bilized and hence it is possible to perform accurate control of the flow rate of cooling medium and to prevent the production of noise due to the vibration of the valve element 8.

[0046] Furthermore, as the ring-shaped portion 11b has no slit in the support ring 10b of this example, this produces the effect that when a large number of support rings 10b are packaged or in an automatic mounting process of expansion valves, the support rings 10b do not intertwine with each other and the automatic mounting process is smoothly performed.

[0047] A fourth example of the support ring will be described by referring to FIGS. 8 to 10.

[0048] As shown in FIG. 8, a support ring 10c in this example comprises one annular ring-shaped portion 11c and three vibration-isolating springs 12a of plate disposed on one side of this ring-shaped portion 11c. In this support ring 10c, an overlapping portion is also formed at the end of the plate forming the ring-shaped portion 11c, in the same manner as in the case of the support ring 10b in the above-described third example.

[0049] This overlapping portion is formed by extending a tongue 11c' having a narrow width and a prescribed length from one end of the ring-shaped portion 11c with the same curvature as the ring-shaped portion 11c. On the other hand, the other end of the ring-shaped portion 11c is formed with a narrow width so as to overlap in the same plane as a tongue 11c'. Incidentally, the shape, material and number of the vibration-isolating springs 12c are the same as those of the support ring 10b of the above-described third example.

[0050] According to the support ring 10c of this structure, the valve element 8 is supported, as shown in FIG. 10, by the vibration-isolating springs 12c at three places in the circumference when this support ring 10c is fixedly attached in the housing space formed in the valve body 5a. That is, this support ring 10c functions as the constraining means of the valve element 8. Therefore, even when fluctuations in the cooling medium pressure occur in the refrigerating cycle, the operation of the valve element 8 can be stabilized and hence it is possible to perform accurate control of the flow rate of cooling medium and to prevent the production of noise due to the vibration of the valve element 8.

[0051] Although in each of the above-described examples of support ring the vibration-isolating springs 12, 12a, 12b, 12c are formed so as to have the same width along their full length, other shapes may be adopted and it is needless to say that elasticity may be adjusted by forming the vibration-isolating springs in such a manner that the vibration-isolating springs take on a triangular shape in which the leading end portion becomes an apex.

[0052] Furthermore, although the slit 13, 13a formed in the ring-shaped portion 11, 11b of the support ring in the first and second examples is formed so as to vertically cross the support ring 10, 10a with respect to the circumferential direction thereof, the slit 13, 13a may be

formed inclined with respect to the circumferential direction of the support ring 10, 10a.

[0053] Furthermore, it is needless to say that the overlapping portion formed at the end of the plate that forms the ring-shaped portion 11b, 11c of the support ring in the third and fourth examples may take on shapes other than those shown in the drawings.

[0054] As is apparent from the above-described descriptions, in the expansion valve according to the present invention, which is provided with the above-described components, it is possible to suppress the vibration of the valve element of expansion valve associated with the pressure fluctuations of a cooling medium. Furthermore, as the constraining means provided in the expansion valve is simple in construction and can be easily worked and it is also easy to mount the constraining means in the valve body, it is possible to realize an expansion valve that is easy to handle and very useful.

Claims

1. An expansion valve in which a valve element adjusts the flow rate of low-pressure cooling medium flowing into an evaporator, said valve element being driven by a temperature-sensing drive section that operates in response to the temperature and pressure of the low-pressure cooling medium fed from the evaporator, said expansion valve comprising:

means for constraining said valve element by giving a constraining force thereto.

2. An expansion valve comprising:

a valve body having an orifice that provides communication between a high-pressure side passage through which a cooling medium flows in and a low-pressure side passage through which the cooling medium flows out;
a valve element that adjusts the volume of the cooling medium flowing through said orifice;
an operating rod that operates said valve element in the valve opening direction; and
a temperature-sensing drive section that drives said operating rod,

wherein constraining means for constraining said valve element is disposed on the upstream side of the orifice of said high-pressure side passage.

3. The expansion valve according to claim 1 or 2, wherein said constraining means is attached to said valve body.
4. The expansion valve according to claim 1 or 2, wherein said constraining means gives a constrain-

ing force to the valve element by an elastic force.

5. The expansion valve according to claim 1 or 2, wherein said valve element is formed in the shape of a ball and the constraining means is a support ring that supports the valve element.

6. The expansion valve according to claim 5, wherein said support ring comprises an elastically deformable, annular ring-shaped portion and a plurality of vibration-isolating springs and said vibration-isolating spring supports the valve element.

7. The expansion valve according to claim 5, wherein said support ring comprises upper and lower annular ring-shaped portions and a plurality of plate-like vibration-isolating springs formed by cutting said ring-shaped portions so as to protrude therefrom.

8. The expansion valve according to claim 5, wherein said support ring comprises one annular ring-shaped portion and a plurality of plate-like vibration-isolating springs disposed on one side of said ring-shaped portion.

9. The expansion valve according to claim 6, wherein said vibration-isolating spring is formed from a curved plate and the valve element is supported on the surface of the curved plate.

10. An expansion valve comprising:

a valve body having an orifice that provides communication between a high-pressure side passage through which a cooling medium flows in and a low-pressure side passage through which the cooling medium flows out;
a valve element that adjusts the volume of the cooling medium flowing through said orifice;
an operating rod that operates said valve element in the valve opening direction;
a temperature-sensing drive section that drives said operating rod; and
a support member that supports said valve element,

wherein constraining means for constraining said support member is disposed on the upstream side of the orifice of said high-pressure side passage.

11. The expansion valve according to claim 10, wherein said valve element is formed in the shape of a ball and said constraining means is a support ring that supports the valve element or/and the support member.

12. The expansion valve according to claim 11, wherein

said support ring comprises an elastically deformable, annular ring-shaped portion and a vibration-isolating spring and said vibration-isolating spring supports the valve element.

5

13. The expansion valve according to claim 11, wherein said support ring comprises upper and lower annular ring-shaped portions and plate-like vibration-isolating springs formed by cutting said ring-shaped portions so as to protrude therefrom.

10

14. The expansion valve according to claim 11, wherein said support ring comprises one annular ring-shaped portion and a plurality of plate-like vibration-isolating spring disposed on one side of said ring-shaped portion.

15

15. The expansion valve according to any one of claim 12, wherein said vibration-isolating spring is formed from a curved plate and the valve element is supported on the surface of the curved plate.

20

16. The expansion valve according to claim 5, wherein said support ring comprises a ring-shaped member made of a metallic elastic material and a slit or an overlapping portion, which enables the diametrical length thereof to be changed, is formed in said ring-shaped member.

25

30

35

40

45

50

55

FIG.1

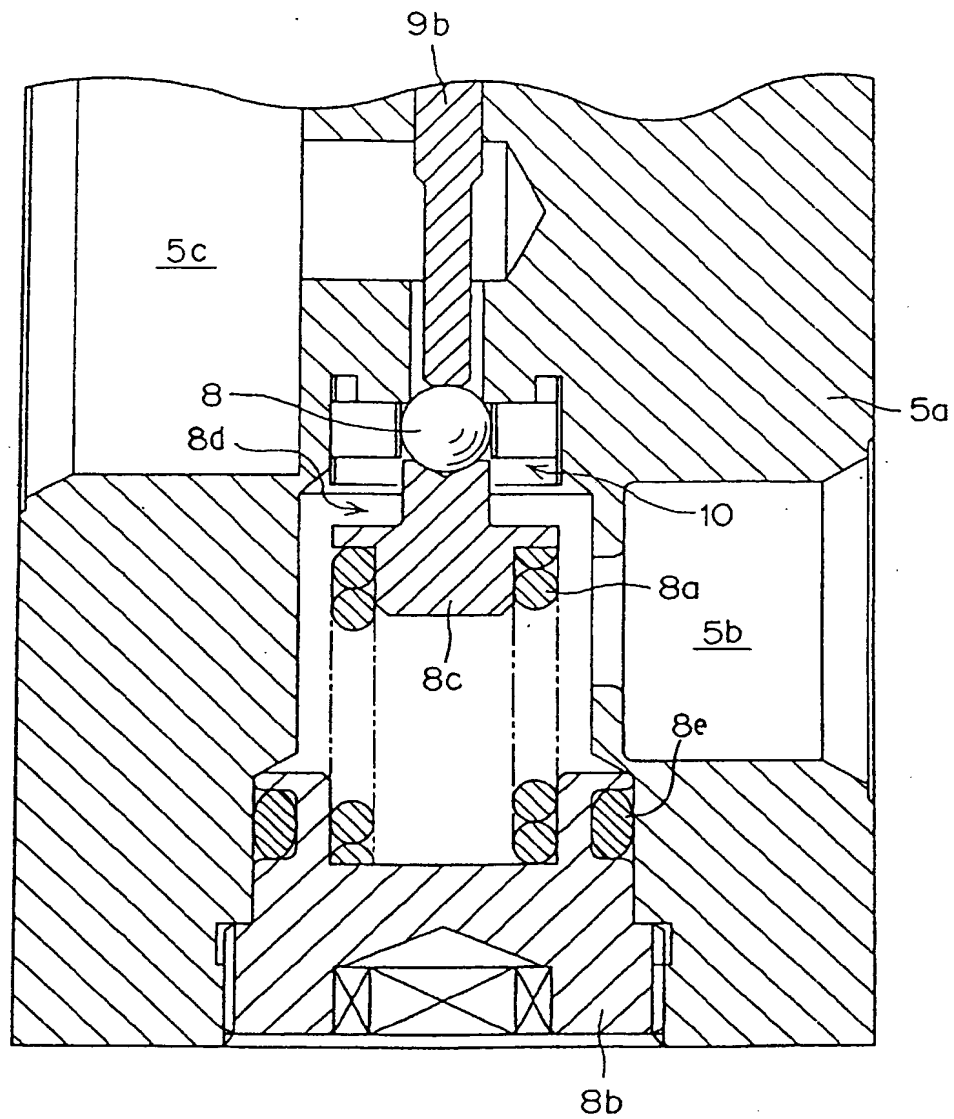


FIG.2

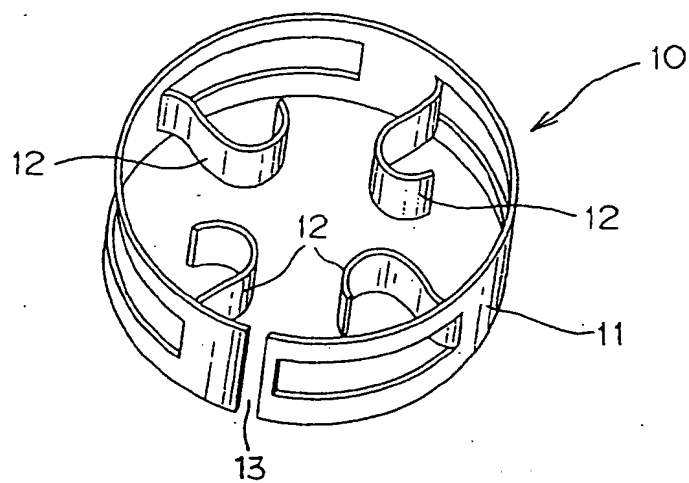


FIG.3

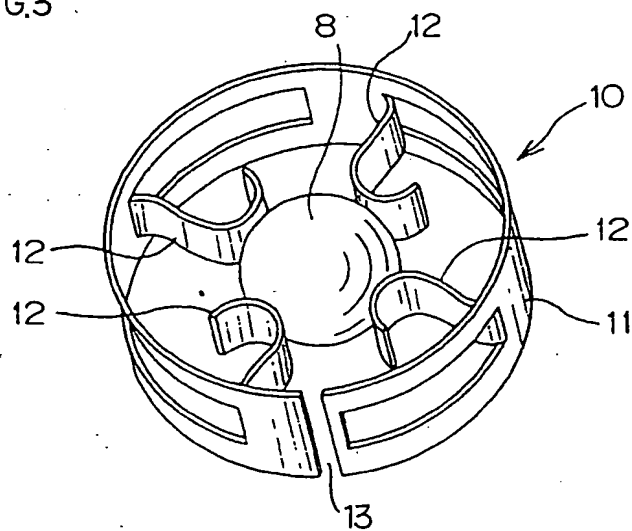


FIG.4

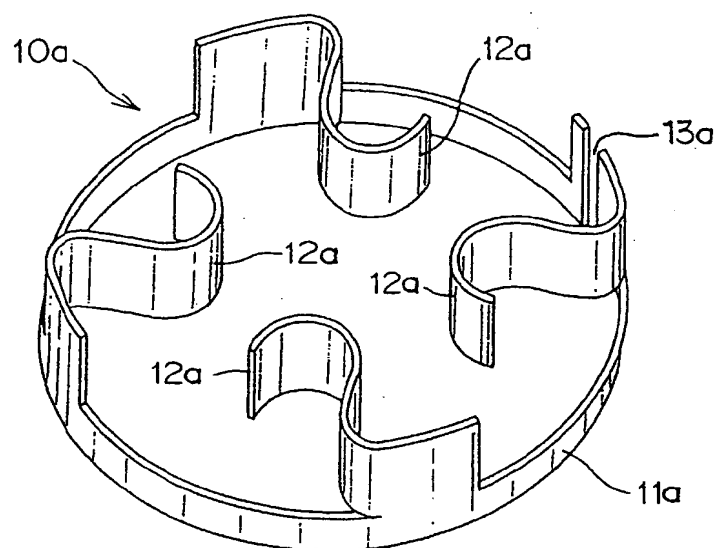


FIG.5

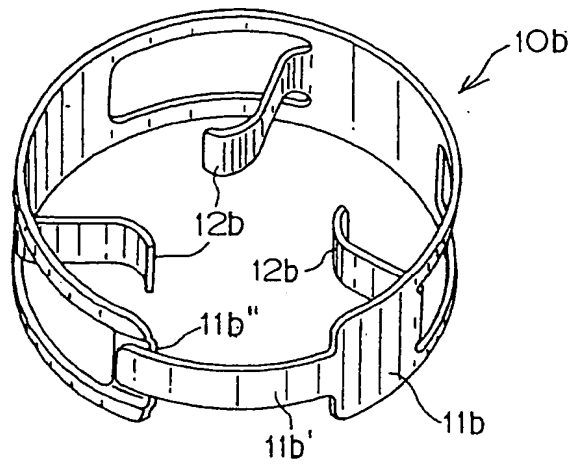


FIG.6

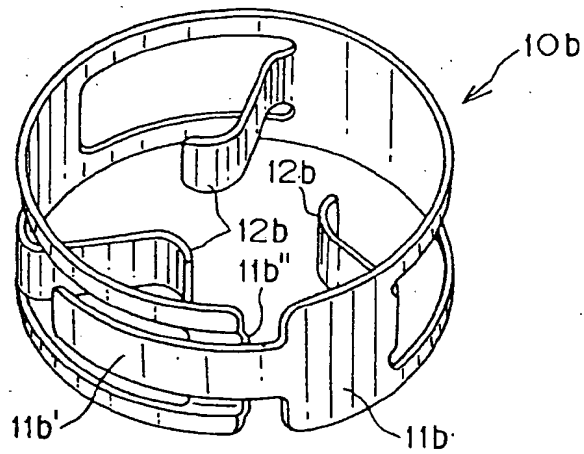


FIG.7

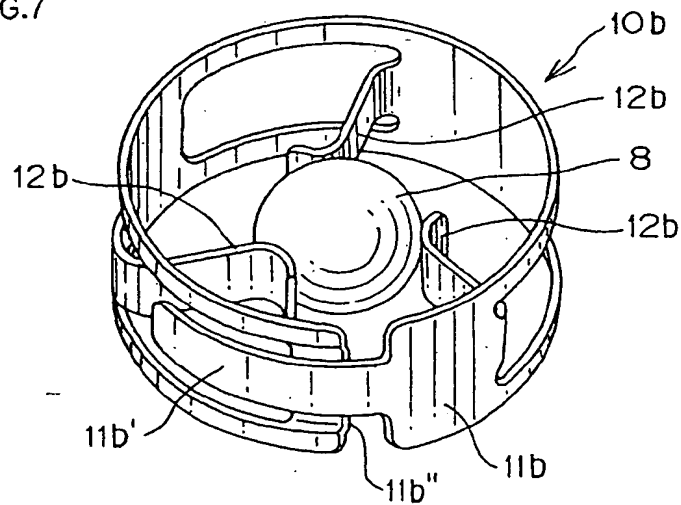


FIG.8

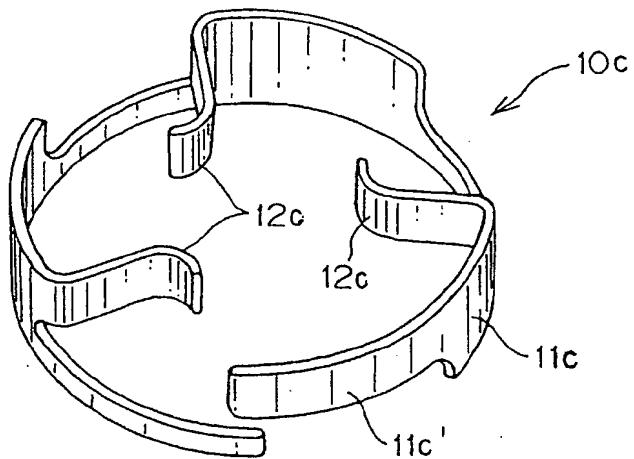


FIG.9

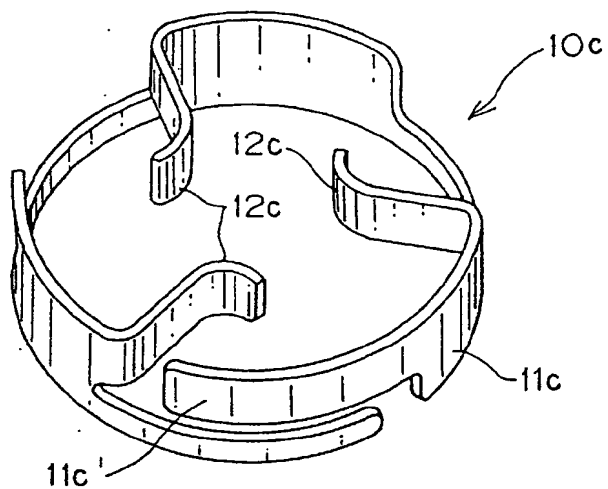


FIG.10

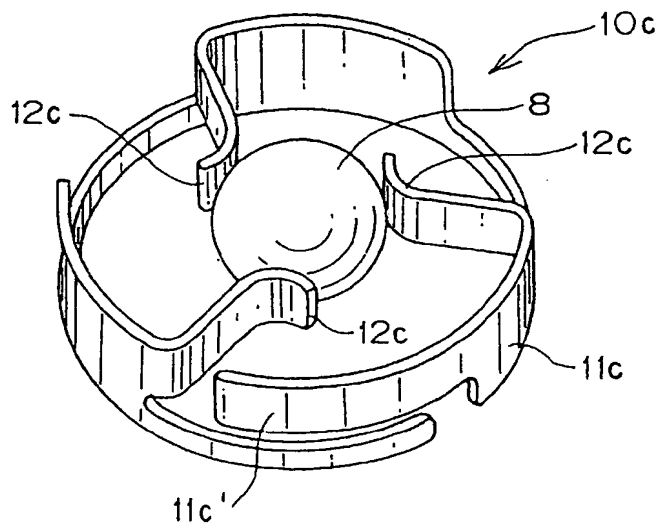
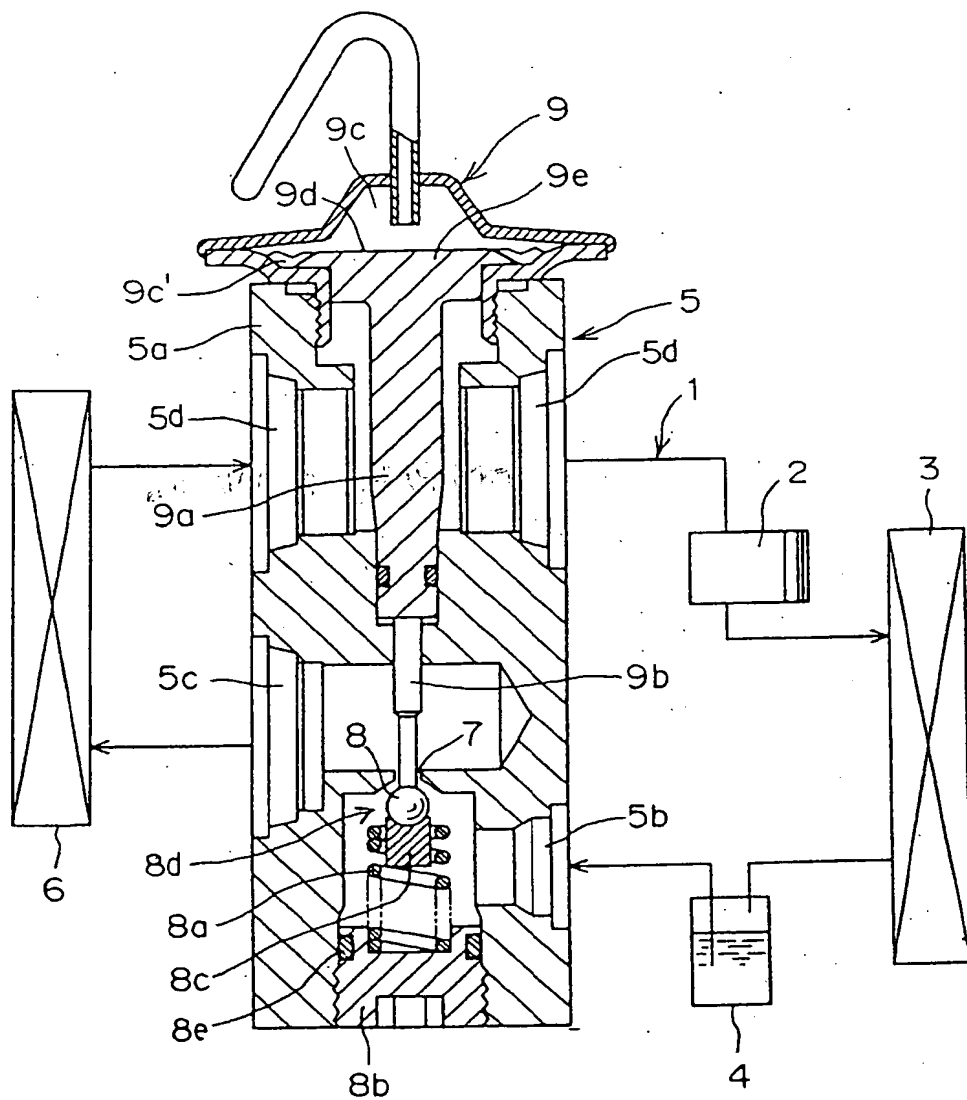


FIG.11



THIS PAGE BLANK (USPTO)

(19)



Europäisches Patentamt
European Patent Office
Office européen des brevets



(11)

EP 1 275 916 A3

(12)

EUROPEAN PATENT APPLICATION

(88) Date of publication A3:
24.09.2003 Bulletin 2003/39

(51) Int Cl.7: **F25B 41/06**, **F16K 31/68**,
G05D 23/12

(43) Date of publication A2:
15.01.2003 Bulletin 2003/03

(21) Application number: **02254796.2**

(22) Date of filing: **09.07.2002**

(84) Designated Contracting States:
AT BE BG CH CY CZ DE DK EE ES FI FR GB GR
IE IT LI LU MC NL PT SE SK TR
Designated Extension States:
AL LT LV MK RO SI

(72) Inventors:
• **Yano, Masamichi**, Fujikoki Corporation
Tokyo 158-0082 (JP)
• **Watarai, Daisuke**, Fujikoki Corporation
Tokyo 158-0082 (JP)

(30) Priority: **12.07.2001 JP 2001211690**
28.12.2001 JP 2001400573

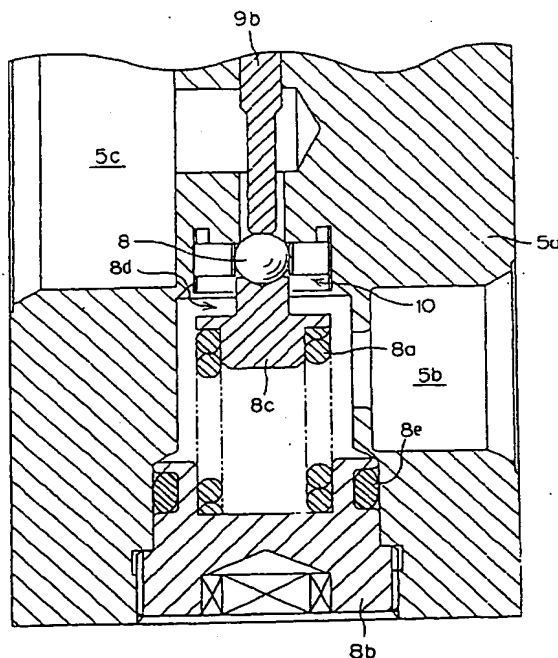
(74) Representative: **Mouteney, Simon James**
MARKS & CLERK,
57-60 Lincoln's Inn Fields
London WC2A 3LS (GB)

(71) Applicant: **Fujikoki Corporation**
Tokyo 158-0082 (JP)

(54) **Expansion valve**

(57) The valve body (5a) has an orifice (7) that provides communication between a high-pressure side passage (5b) through which a cooling medium flows in and a low-pressure side passage (5c) through which the cooling medium flows out. Also, the valve (5) is provided with a valve element (8) that adjusts the volume of the cooling medium passing through this orifice, an operating rod (9b) that operates the valve element (8) in the valve opening position, and a temperature-sensing drive section (9) that drives this operating rod (9b). On the upstream side of the orifice (7) of the high-pressure side passage (5b) is disposed a support ring (10) that constrains the ball-shaped valve element (8) of the valve body (5a).

FIG.1



EP 1 275 916 A3



European Patent
Office

EUROPEAN SEARCH REPORT

Application Number
EP 02 25 4796

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (InCL.7)
A	US 6 145 753 A (WATANABE KAZUHIKO ET AL) 14 November 2000 (2000-11-14) * column 4, line 53 - column 8, line 9; figures 1-6 *	1,2,10	F25B41/06 F16K31/68 G05D23/12
A	US 4 542 852 A (TORRENCE ROBERT J ET AL) 24 September 1985 (1985-09-24) * column 2, line 31 - column 4, line 1; figures 4-6 *	1,2,10	
A	GB 540 730 A (CYRIL ALPHONSO PUGH; JOHN BROWN GIBSON; PLESSEY CO LTD) 28 October 1941 (1941-10-28) * page 3, line 18 - line 75; figure *	1-4,10	
A	PATENT ABSTRACTS OF JAPAN vol. 2000, no. 20, 10 July 2001 (2001-07-10) -& JP 2001 082835 A (DENSO CORP), 30 March 2001 (2001-03-30) * abstract; figures 4,6 *		
A	US 6 029 694 A (HOEPFL ROLAND ET AL) 29 February 2000 (2000-02-29)		F25B F16K
The present search report has been drawn up for all claims			
Place of search THE HAGUE		Date of completion of the search 5 August 2003	Examiner Boets, A
<p>CATEGORY OF CITED DOCUMENTS</p> <p>X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document</p> <p>T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons & : member of the same patent family, corresponding document</p>			

EPO FORM 1503 03 82 (P04C01)

**ANNEX TO THE EUROPEAN SEARCH REPORT
ON EUROPEAN PATENT APPLICATION NO.**

EP 02 25 4796

This annex lists the patent family members relating to the patent documents cited in the above-mentioned European search report. The members are as contained in the European Patent Office EDP file on
The European Patent Office is in no way liable for these particulars which are merely given for the purpose of information.

05-08-2003

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
US 6145753 A	14-11-2000	JP 10253199 A	25-09-1998
		CN 1193094 A	16-09-1998
		DE 69715809 D1	31-10-2002
		EP 0864826 A2	16-09-1998
US 4542852 A	24-09-1985	AU 3793485 A	12-09-1985
		BR 8501030 A	29-10-1985
		DE 3507221 A1	05-09-1985
		FR 2560662 A1	06-09-1985
		IT 1183337 B	22-10-1987
GB 540730 A	28-10-1941	NONE	
JP 2001082835 A	30-03-2001	NONE	
US 6029694 A	29-02-2000	DE 19649554 A1	04-06-1998
		FR 2756604 A1	05-06-1998
		JP 10171538 A	26-06-1998

EPO FORM P0459

For more details about this annex : see Official Journal of the European Patent Office, No. 12/82

THIS PAGE BLANK (USPTO)